

LAMINAR THERMAL MIXING IN PLANE SHEAR FLOW

A. Haas¹, M. Scholle¹, H.M. Thompson², R.W. Hewson², N. Aksel¹, P.H. Gaskell²

¹University of Bayreuth
Department of Applied Mechanics and Fluid Dynamics
D-95440, Bayreuth, Germany

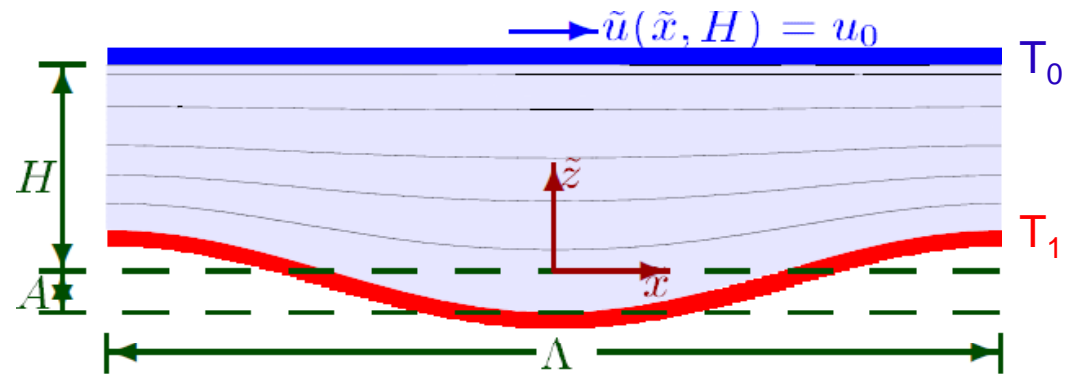
²University of Leeds
Engineering Fluid Mechanics Research Group
School of Mechanical Engineering
Leeds, LS2 9JT, UK

Problem formulation

- Two-dimensional
- Steady
- Periodic BC
- Sinusoidally corrugated substrate
- Constant λ , c_p and ρ

Parameters:

- Wavelength: Λ
- Amplitude: A
- Mean gap width: H
- Lid velocity: u_0
- Temperature: $T_1 > T_0$
- Viscosity: $\eta(T) = \eta_0 (1 - \eta^* T)$



Non-dimensional model equations

Simplifications: **No buoyancy and no dissipation**

Continuity equation: $u_x + w_z = 0$

Navier-Stokes equations:

$$\text{Re} [uu_x + ww_z] = -p_x + 2\partial_x [(1 - \eta^* T)u_x] + \partial_z [(1 - \eta^* T)(u_z + w_x)]$$

$$\text{Re} [uw_x + ww_z] = -p_z + 2\partial_z [(1 - \eta^* T)w_z] + \partial_x [(1 - \eta^* T)(u_z + w_x)]$$

Temperature equation: $\text{Pe} [uT_x + wT_z] - [T_{xx} + T_{zz}] = 0$

Dimensionless numbers:

$$\text{Re} = \frac{\rho_0 u_0 \Lambda}{2\pi\eta_0} \quad \text{Pe} = \frac{\Lambda u_0 \rho_0 c_p}{2\pi\lambda}$$

- **Semi-analytical approach**
 - Based on a variational formulation
 - Stokes flow limit $Re \rightarrow 0$
 - neglect of thermoviscosity coupling term

- **Finite element methods**

Comsol® Multiphysics modules

 - Incompressible Navier-Stokes
 - Heat transfer/convection

Unilaterally coupled set of equations:

- Hydrodynamic field:

$$\begin{aligned}u_x + w_z &= 0 & u(x, -a \cos x) &= 0 \\-p_x + u_{xx} + u_{zz} &= 0 & w(x, -a \cos x) &= 0 \\-p_z + w_{xx} + w_{zz} &= 0 & u(x, h) &= 1 \\ & & w(x, h) &= 0\end{aligned}$$

- Temperature field

$$\begin{aligned}\text{Pe} [uT_x + wT_z] - T_{xx} - T_{zz} &= 0 & T(x, -a \cos x) &= 1 \\ & & T(x, h) &= 0\end{aligned}$$

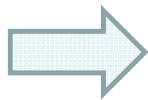
a: non-dimensional amplitude

h: non-dimensional gap-width

Decoupling:

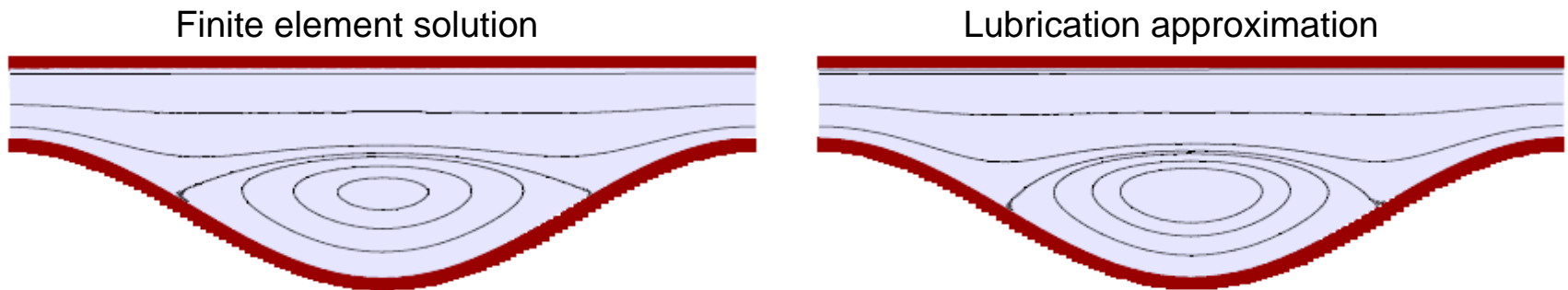
1) Isothermal flow

2) Heat conduction with convection



Lubrication approximation

➔ Analytic solution for the assumption $a \ll 1$



➔ Reasonable results even for moderate amplitudes $a < 1$

Stokes flow - Heat conduction & convection

Variational formulation: $\delta I = 0$ with **non-local** functional:

$$I := \int_{-\pi - a \cos x}^{\pi} \int_{-a \cos x}^h \left[-\text{Pe} T(x, z) (u \cdot \nabla) T(-x, z) + \nabla T(x, z) \cdot \nabla T(-x, z) \right] dz dx$$

Reproduces the heat conduction equation with convection:

$$\text{Pe} [u T_x + w T_z] - T_{xx} - T_{zz} = 0$$

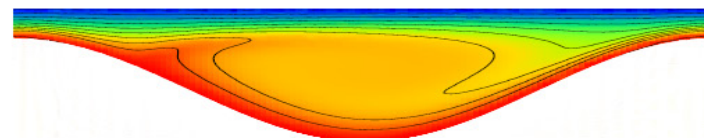
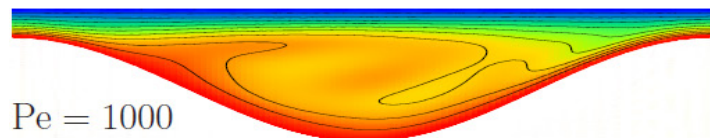
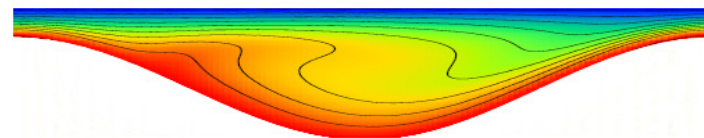
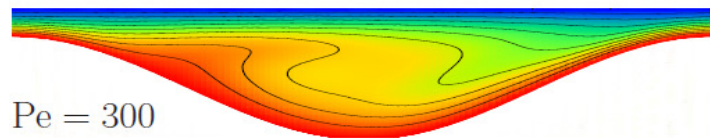
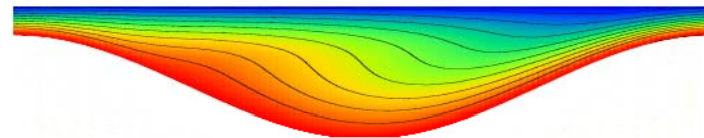
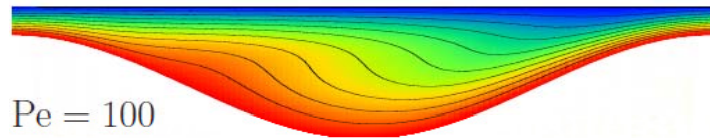
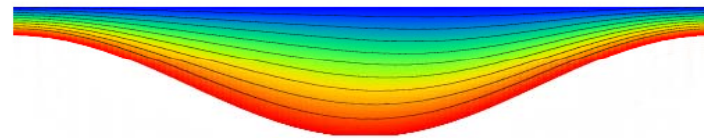
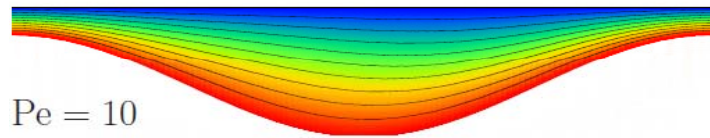
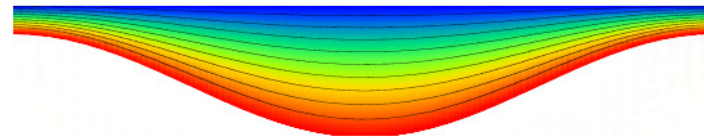
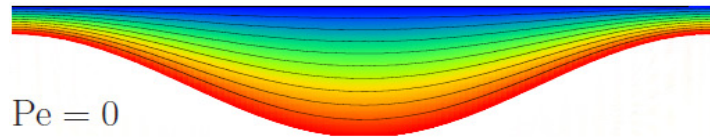


Solution by Ritz's direct method with appropriate base functions.

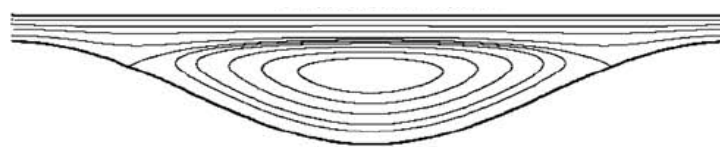
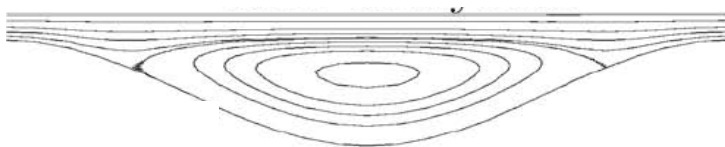
Stokes flow - Temperature field

Semi-analytical solution

Finite element solution



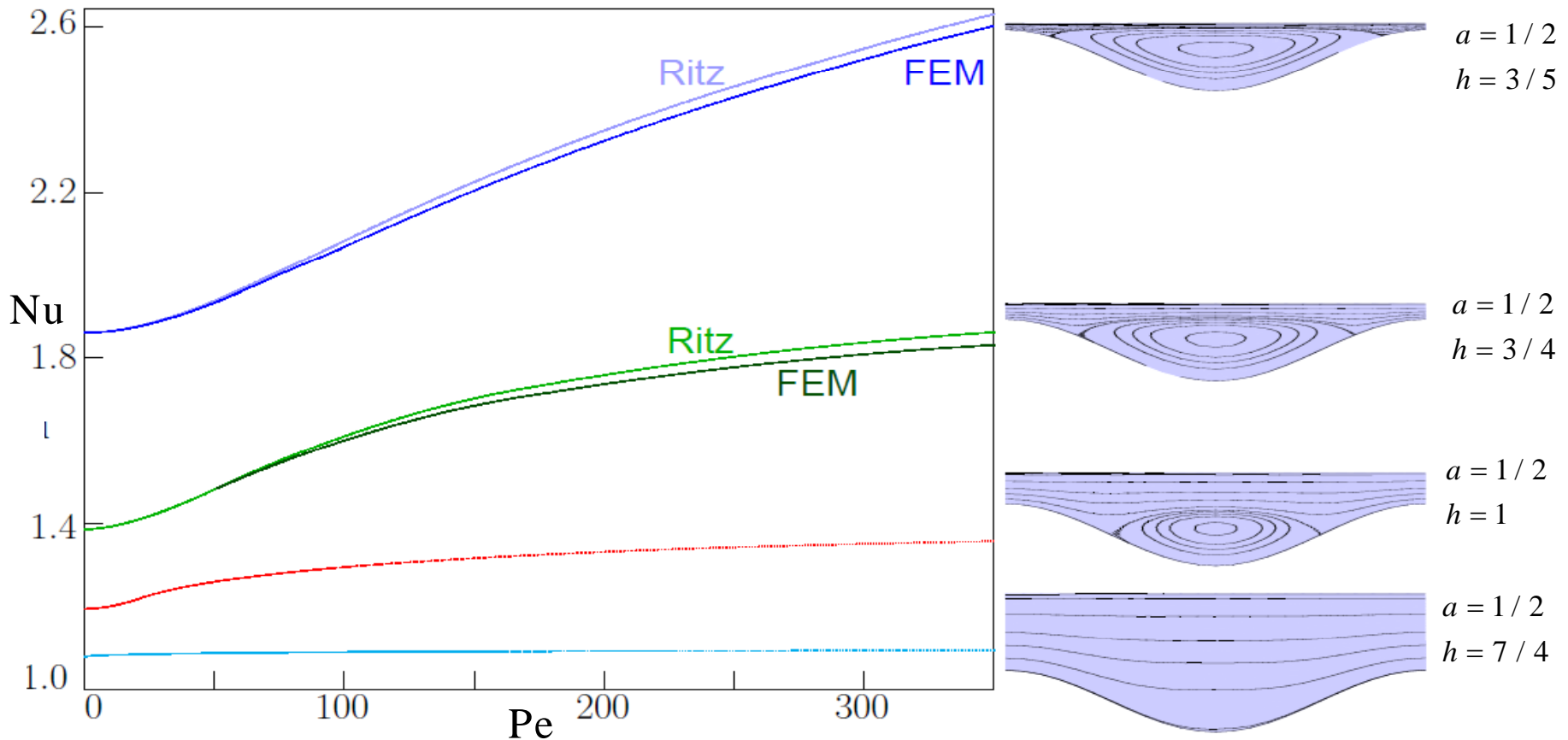
Streamlines:



$$a = 1/2$$
$$h = 3/4$$

Stokes flow - Global heat transfer

Nusselt number $Nu = \frac{h\dot{Q}}{\Lambda(T_2 - T_1)}$ vs. Peclét number $Pe = \frac{\rho c_p u_0 \lambda}{2\pi\Lambda}$

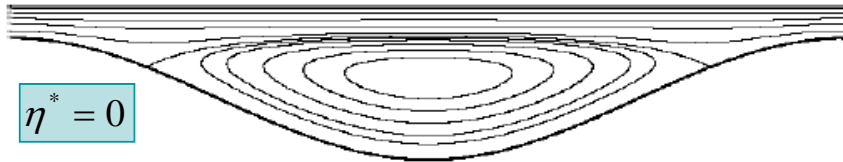


Stokes flow - Thermal feedback

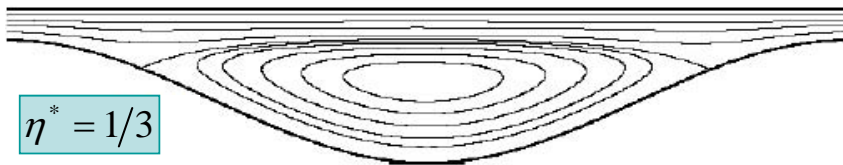
Thermoviscosity:

$$\eta(T) = \eta_0 (1 - \eta^* T)$$

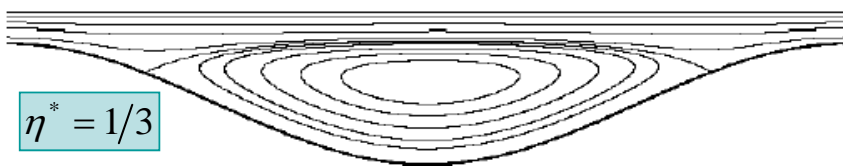
Hydrodynamic streamlines:



$$\eta^* = 0$$

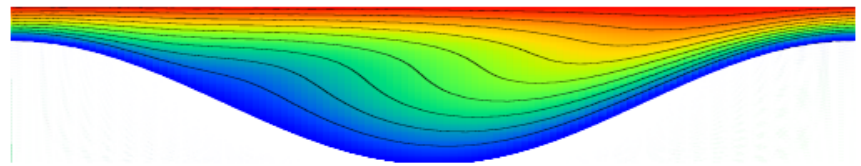
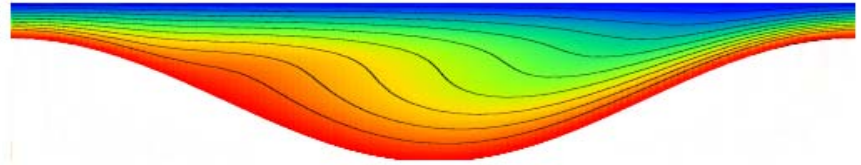
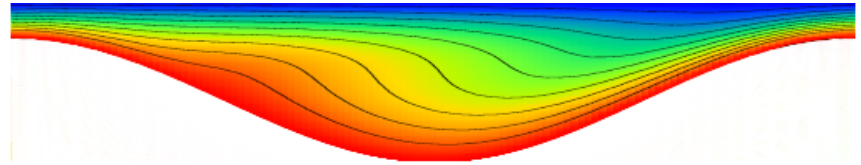


$$\eta^* = 1/3$$



$$\eta^* = 1/3$$

Temperature field:



$$a = 1/2$$

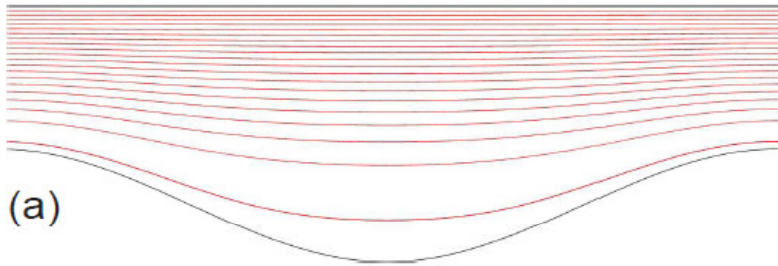
$$h = 3/4$$

$$\text{Pe} = 100$$

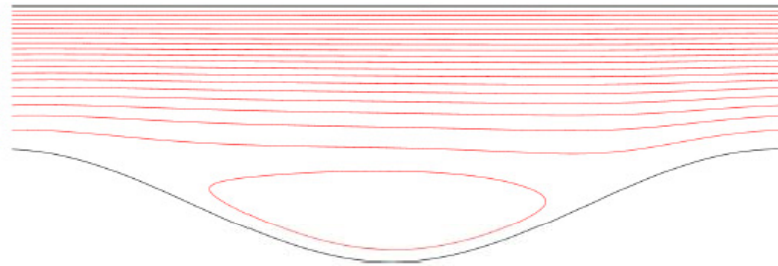
Inertial effects – Flow structure

Re \rightarrow 0

Re = 100

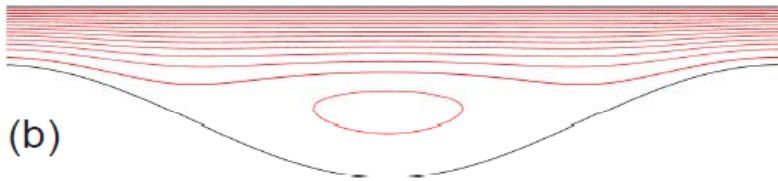


(a)

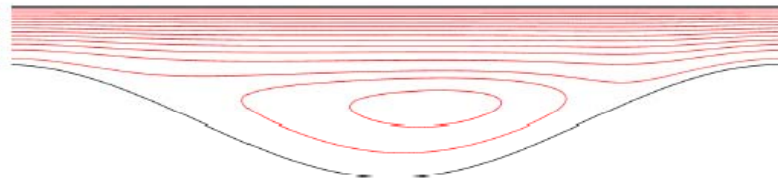


$$a = 1/2$$

$$h = 7/4$$

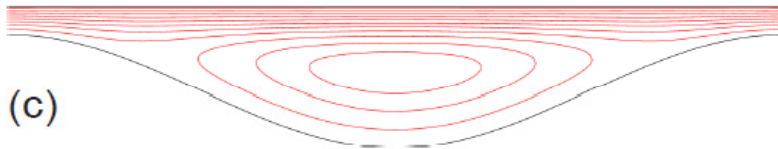


(b)

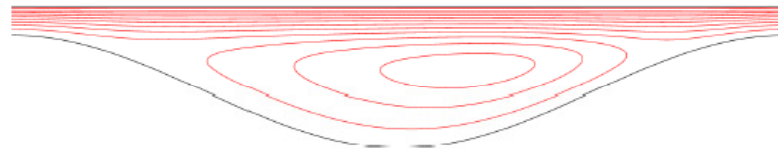


$$a = 1/2$$

$$h = 1$$

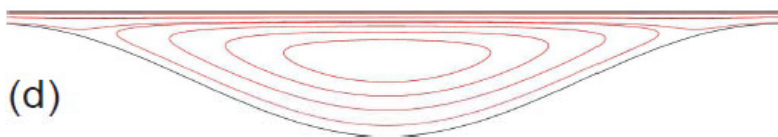


(c)

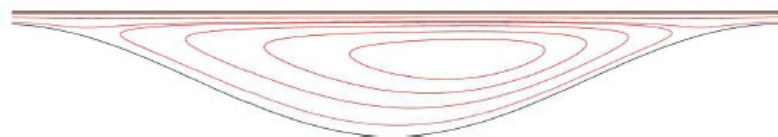


$$a = 1/2$$

$$h = 3/4$$



(d)

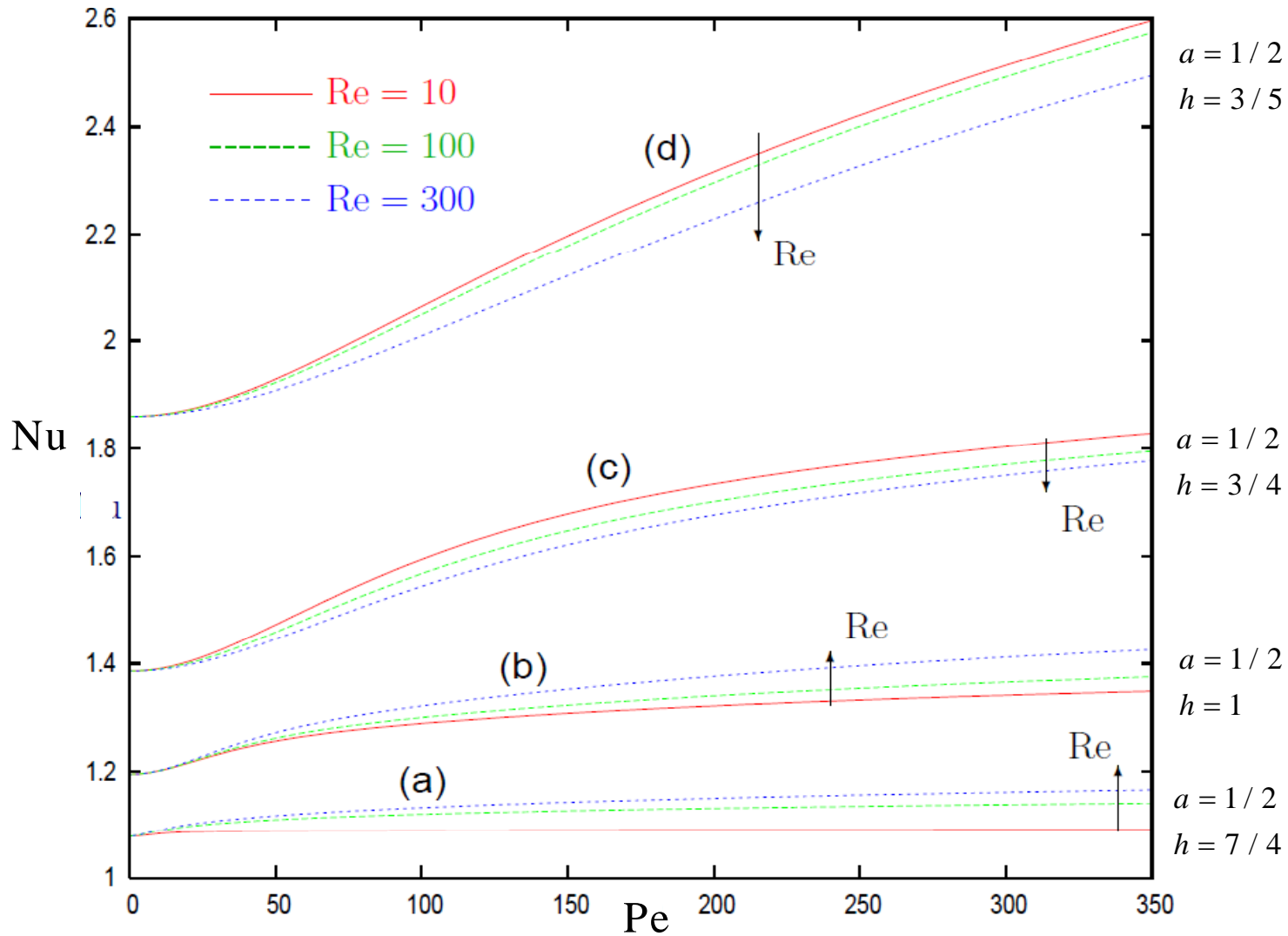


$$a = 1/2$$

$$h = 3/5$$

Pe = 100

Inertial Effects – Global Heat Transport



- Stokes flow limit: $Re \rightarrow 0$
 - Global Heat transfer: Comparison between semi-analytical and finite element solution
 - Influence of thermoviscosity
- Inertial effects on global heat transfer
 - Competing kinematical and inertial effects.

Available literature:

M. Scholle, A. Haas, R.W. Hewson, H.M. Thompson, P.H. Gaskell, N. Aksel, *The effect of locally induced flow structure on global heat transfer for plane laminar shear flow*. Int. J. Heat & Fluid Flow **30**, 175-185 (2009).

M. Scholle, A. Haas, M.C.T. Wilson, H.M. Thompson, P.H. Gaskell, N. Aksel, *Eddy genesis and manipulation in plane shear flow*. Phys. Fluids **21**, 073602 (2009).